

YDAC INTERNATIONAL



Hydraulic dampers

1. DESCRIPTION

1.1. FUNCTION

The pressure fluctuations occurring in hydraulic systems can be cyclical or one-off

- problems due to:

 flow rate fluctuations from displacement pumps
- actuation of shut-off and control valves with short opening and closing times
- switching on and off of pumps
- sudden linking of spaces with different pressure levels.

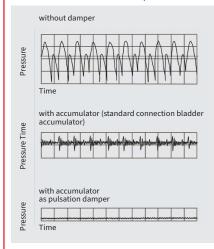
HYDAC hydraulic dampers are particularly suitable for damping such pressure fluctuations.

Selecting the most suitable hydraulic damper for each system ensures that

- vibrations caused by pipes, valves, couplings etc. are minimised and subsequent pipe and valve damage is prevented
- measuring instruments are protected and their performance is no longer impaired
- the noise level in hydraulic systems is reduced
- the performance of machine tools is improved
- interconnection of several pumps in one line is possible
- a pump rpm and feed pressure increase is possible
- the maintenance and servicing costs can be reduced
- the service life of the system is increased.

2. APPLICATION

2.1. PULSATION DAMPING TYPE SB...P / SBO...P



2.1.1 General The HYDAC pulsation damper

- prevents pipe breaks caused by material fatigue, pipe oscillations and irregular
- protects valves, control devices and other instruments,
- improves noise level damping.

2.1.2 Applications

flow rates.

The pulsation damper is particularly suitable for hydraulic systems, displacement pumps, sensitive measurement and control instruments and manifolds, e.g. in process circuits in the chemical industry.

2.1.3 Mode of operation

The pulsation damper generally has two fluid connections and can therefore be fitted directly inline.

The flow is diverted in the fluid valve so the tbilaid dierecot etilap traight. This causes direct contact

of the flow with the bladder or diaphragm which, in an almost inertia-less operation, balances the flow rate fluctuations via the gas volume.

It particularly compensates for higher frequency pressure oscillations. The charge pressure is adjusted to individual operating conditions.

2.1.4 Design The HYDAC pulsation damper consists of:

- the welded or forged pressure vessel in carbon steel; available with internal coating or in stainless steel for chemically aggressive fluids.
- the special fluid valve with inline connection, which guides the flow into the vessel (threaded or flange connection).
- the bladder or diaphragm in various elastomers as shown under section 4.2.

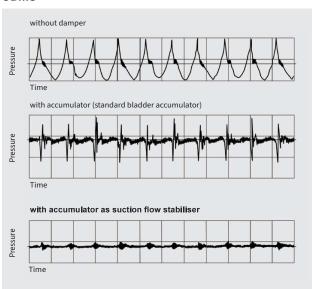
2.1.5 Installation

As close as possible to the pulsation source. Mounting position preferably vertical (gas valve pointing upwards).

Preferred and alternative installation positions are shown in schematic form in . section 3.



2.2. SUCTION FLOW STABILISER TYPE SB...S



2.2.1 General

The HYDAC suction flow stabiliser

- improves the NPSH value of the system
- prevents cavitation of the pump
- prevents pipe oscillations.

2.2.2 Applications

Main application areas are piston and diaphragm pumps in public utility plants, reactor construction and the chemical industry.

2.2.3 Mode of operation

Trouble-free pump operation is only possible if no cavitation occurs

in the pump suction and pipe oscillations are prevented. A relatively high fluid volume in the suction flow stabiliser in relation to the displacement volume of the pump reduces the acceleration effects of the fluid column in the suction line. Also an air separation is achieved due to the extremely low flow rate in the By cations flowy to the bilisseering points and a file to the blank to the base operating conditions, the best possible pulsation damping is achieved.

2.2.4 Design

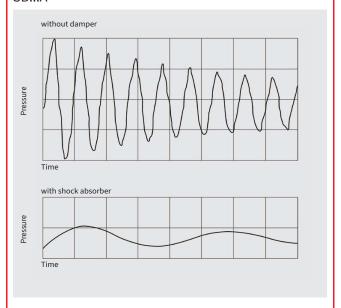
The HYDAC suction flow stabiliser consists of a welded vessel in steel or stainless steel.

Inlet and outlet are on opposite sides and are separated by Tallscaffleper part houses the encapsulated bladder. In addition, there is an air bleed screw in the cover and a drainage facility on the bottom.

2.2.5 Installation

As close as possible to the suction inlet of the pump. Mounting position vertical (gas valve pointing upwards).

2.3. SHOCK ABSORBER TYPE SB...A



2.3.1 General

The HYDAC shock absorber

- reduces pressure shocks
- protects pipelines and valves from being destroyed.

The accumulators are particularly suitable for use in pipelines with quick-acting valves or flaps and whilst pumps are being switched on and off

They are also suitable for energy storage in low pressure applications.

2.3.3 Mode of operation

Sudden changes in pipeline flow, such as those caused by pump failure or the closing or opening of valves, can cause pressures which are many times higher than the normal values.

The shock absorber prevents this by converting potential into kinetic energy and vice versa. This prevents pressure shocks and protects pipelines, valves, monitoring instruments and other pipe fittings from destruction.

2.3.4 Design The HYDAC shock absorber consists of:

- the welded pressure vessel in carbon steel with or without corrosion protection or in stainless steel.
- the connection including perforated disc which prevents the flexible bladder from extruding from the vessel, and the flange.
- the bladder in various elastomer qualities as shown under section 4.2 with built-in gas valve, which is used for charging pressure p₀ and for possible monitoring activities.

2.3.5 Special version

Shock absorbers can also be in the form of diaphragm or piston accumulators. Available on request.

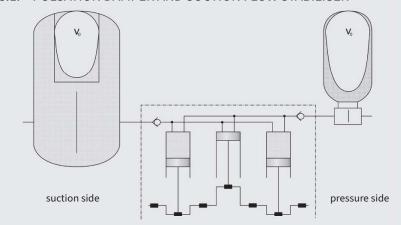
2.3.6 Installation

As close as possible to the source of the erratic condition. Mounting position vertical (gas valve pointing upwards).



3 SIZING

3.1. PULSATION DAMPER AND SUCTION FLOW STABILISER



On the suction and the pressure side of piston pumps almost identical conditions occur regarding irregularity of the flow rate. Therefore the same formulae for determining the effective gas volume are used for calculating the damper size. That in the end two totally different damper types are used is due to the different acceleration and pressure ratios on the two sides.

Not only is the gas volume V_0 a decisive factor but also the connection size of the pump has to be taken into account when selecting the pulsation damper. In order to avoid additional variations in cross-section, which represent reflection points for vibrations, and also to keep pressure drop to a reasonable level, the fitting cross-section of the damper must be the same as that of the pipeline.

The gas volume V_0 of the damper is determined with the aid of the formula for adiabatic changes of state.

By giving the residual pulsation or the gas volume, the damper size can be dimensioned with the aid of the HYDAC software ASP (Accumulator Simulation Program).

Designations:

ΔV = fluctuating fluid volume [I]

q = stroke volume [l]

$$q = \frac{\pi \cdot d_{\kappa}^2}{4} \cdot h_{\kappa}$$

d_k= piston diameter [dm]

h_k= piston stroke [dm]

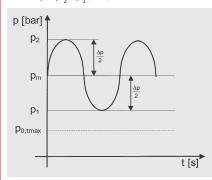
$$m = \Delta V$$

- z = no. of compressions/ effective cylinders per revolution
- $x = residual pulsation [\pm \%]$
- κ = isentropic exponent
- Φ = pressure ratio of pre-charge pressure to operating pressure [0.6 ... 0.9]

$$\Phi = \frac{p_0}{p_m}$$

$$\Delta p = cyclic test pressure$$

 $\Delta p = p_2 - p_1 [bar]$



Formulae

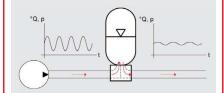
$$V_0 = \frac{\Delta V}{\left[\frac{\Phi}{1 - \frac{X}{100}}\right]^{\frac{1}{\kappa}} - \left[\frac{\Phi}{1 + \frac{X}{100}}\right]^{\frac{1}{\kappa}}}$$

$$\Delta V = m \cdot q$$

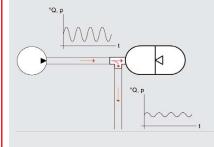
$$x [\pm \%] = \left| \frac{p_1 - p_m}{p_m} \cdot 100 \right|$$

Diagram of mounting options:

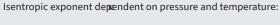
Preferred installation configuration with maximum damping effect

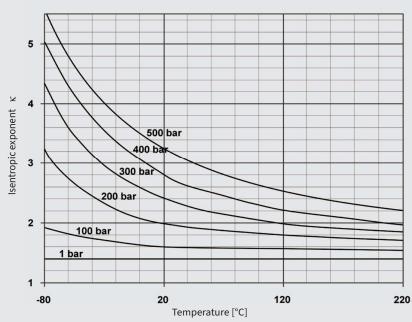


Alternative installation configuration using standard accumulator with a T-piece with reduced damping effect









Amplitude factor (m) for piston pump:

		<u> </u>
	m valu	е
Z	single acting	double acting
1	0.548	0.206
2	0.206	0.042
3	0.035	0.018
5	0.042	0.010
5	0.010	0.007
6	0.018	0.005
7	0.005	
8	0.010	-
9	0.001	
others	on request	

3.1.1 Calculation example Given parameters:

Single-acting 3-piston pump
Piston diameter: 70 mm
Piston stroke: 100 mm
Drive speed: 370 rpm
Flow rate: 427 l/min
Operating temperature: 20 °C
Operating pressure
- pressure side: 200 bar
- suction side: 4 bar

Required:

- a) Suction flow stabiliser for a residual pulsation of ± 2.5%
- b) Pulsation damper for a residual pulsation of ± 0.5%

Solution

a) Determining the required suction flow stabiliser

$$V_0 = \frac{\Delta V}{\left[\frac{\Phi}{1 - \frac{X}{100}}\right]^{\frac{1}{\kappa}} - \left[\frac{\Phi}{1 + \frac{X}{100}}\right]^{\frac{1}{\kappa}}}$$

$$V_0 = \frac{0.035 \cdot \frac{\pi \cdot 0.7^2}{4} \cdot 1.0}{\left[\frac{0.6}{1 - \frac{2.5}{100}}\right]^{\frac{1}{1.4}} - \left[\frac{0.6}{1 + \frac{2.5}{100}}\right]^{\frac{1}{1.4}}}$$

 $V_0 = 0.54 l$

Selected: SB16S-12 with 1 litre gas volume

b) Determining the required pulsation damper

$$V_0 = \frac{\Delta V}{\left[\frac{\Phi}{1 - \frac{x}{100}}\right]^{\frac{1}{\kappa}} - \left[\frac{\Phi}{1 + \frac{x}{100}}\right]^{\frac{1}{\kappa}}}$$

$$V_0 = \frac{0.035 \cdot \frac{\pi \cdot 0.7^2}{4} \cdot 10}{\left[\frac{0.7}{1 - \frac{0.5}{100}}\right]^{\frac{1}{2.0}} - \left[\frac{0.7}{1 + \frac{0.5}{100}}\right]^{\frac{1}{2.0}}}$$

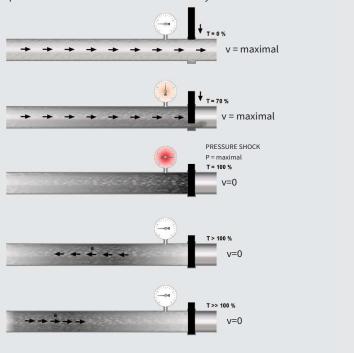
v o= 3.2 l

Selected: SB330P-4



3.2. SHOCK ABSORBER

Pressure shock produced when a valve is closed without a hydraulic accumulator



Simplified pressure shock calculation for the closing of a valve

Estimate of Joukowsky's max.

occurring pressure shock

 $\Delta p[N/m^2] = \rho \cdot a \cdot \Delta v$ $\rho [kg/m^3] = fluid density$

 $\Delta V = V - V_1$

 $\Delta v = \text{change of fluid}$

velocity

v [m/s] = fluid velocity

before the change

in its condition \boldsymbol{v}

[m/s] = fluid velocity

after the change in its condition

a [m/s] = propagation velocity

of pressure wave

$$a [m/s] = \frac{1}{\sqrt{\rho \cdot \left[\frac{1}{K} + \frac{D}{E \cdot e}\right]}}$$

 $K[N/m^2]$ = compression modulus of the fluid

E [N/m²] = module of elasticity of pipeline

D [mm] = internal diameter of the pipeline e [mm] = wall thickness of the pipeline

The pressure wave runs to the other end of the pipeline and will reach the valve again after time t (reflection time), whereby:

L [m] = length of the pipeline

T [s] = eff. operating time

(closing) of the valve
If T < t then:

 $p_{max} = p_1 + \Delta p$

If T > t then:

If I >t then:

$$p_{mx} = t = p_1 + a \vee T \cdot \Delta \cdot -$$

Determining the required damper size
The accumulator must absorb the kinetic
energy of the fluid by converting it into
potential energy within the pre-determined
pressure range. The change of state of the gas
is adiabatic in this case.

$$V_{0} = \frac{m \cdot v^{2} \cdot 0.4}{2 \cdot p_{1} \cdot \left[\left[\frac{p_{2}}{p_{1}} \right]^{1 - \frac{1}{\kappa}} - 1 \right] \cdot 10^{2}} \cdot \left[\frac{p_{1}}{p_{0}} \right]^{\frac{1}{\kappa}}$$

m [kg] = weight of the fluid in the pipeline

v [m/s] = change in velocity of the fluid

 (p_1ar) = zero head of the pump p

[bar] = perm. operating pressure p

[bar] = pre-charge pressure

A special calculation program to analyse the pressure curve is available for dimensioning during pump failure or start-up and for manifolds.



3.2.1 Calculation example Rapid closing of a shut-off valve in a re-fuelling line.

Given parameters:

Length of the pipeline L: 2000 m

Size of pipeline D:

Wall thickness of pipeline e:

6.3 mm

Material of pipeline:

Steel

Flow rate Q: $432 \text{ m}^3/\text{h} = 0.12 \text{ m}^3/\text{s}$

Density of medium ρ : 980 kg/m³

Zero feed height of pump p1: 6

Min. operating pressure pmin:

Eff. closing time of the valve T:

(approx. 20% of total closing time)

Operating temperature:

Compression modulus of the fluid K:

 $1.62\,\times10^{\,9}\,\,\text{N/m}^2$

Module of elasticity (steel) E: 2.04 $\,\times\,10^{\,11}\,\,\text{N/m}^2$

Required:

Size of the required shock absorber, when the max. pressure (p2) must not exceed 10

Solution:

Determination of reflection time:

$$a = \frac{1}{\sqrt{\rho \cdot \left[\frac{1}{K} + \frac{D}{E \cdot e}\right]}}$$

$$a = \frac{1}{\sqrt{\frac{1}{K} + \frac{D}{E \cdot e}}}$$

$$a = \sqrt{980 \cdot \left[\frac{1}{1.62 \cdot 10^9} + \frac{250}{2.04 \cdot 10^{11} \cdot 6.3} \right]}$$

a = 1120 m/s

$$t = \frac{2 \cdot L}{a} = \frac{2 \cdot 2000}{1120} = 3.575 s^*$$

* since T < t the max. pressure surge occurs and the formula as shown in section 3.2. must be used.

$$v = \frac{Q}{A}$$

$$v = \frac{0.12}{0.25^2 \cdot \frac{\pi}{4}} = 2.45 \text{ m/s}$$

$$\Delta_{p} = \rho \cdot a \cdot \Delta v$$

$$\Delta_{p}^{F} = 980 \cdot 1120 \cdot (2.45-0) \cdot 10^{-5}$$
= 26.89 bar

$$p_{max} = p_1 + \Delta_p$$

$$p_{max} = p_1 + \Delta_p$$

 $p_{max} = 6 + 26.89 = 32.89 \text{ bar}$

Determining the required gas volume:

$$p_0 \leq 0.9 \cdot p_{min}$$

$$p_0 \le 0.9 \cdot 5 = 4.5 \text{ bar}$$

$$V_{0} = \frac{m \cdot v^{2} \cdot 0.4}{2 \cdot p_{1} \cdot \left[\left[\frac{p_{2}}{p_{1}} \right]^{1 \cdot \frac{1}{\kappa}} - 1 \right] \cdot 10^{2}} \cdot \left[\frac{p_{1}}{p_{0}} \right]^{\frac{1}{\kappa}}$$

with
$$m = V \cdot \rho = \frac{\pi}{4} \cdot D^2 \cdot L \cdot \rho$$

$$V_0 = \frac{\frac{\pi}{4} \cdot 0.25^2 \cdot 2000 \cdot 980 \cdot 2.45^2 \cdot 0.4}{2 \cdot 7 \cdot \left[\left[\frac{11}{7} \right]^{\frac{1}{1.4}} - 1 \right] \cdot 10^2} \cdot \left[\frac{7}{4.5} \right]^{\frac{1}{1.4}}$$

$$V_0 = 1641 I$$

Selected:

4 x shock absorbers SB35AH-450



4. SPECIFICATIONS

4.1. EXPLANATIONS, NOTES

4.1.1 Operating pressure see tables (may differ from nominal pressure for foreign test certificates).

4.1.2 Nominal volume see

4.1.3 Effective gas volume see tables, based on nominal dimensions. This differs slightly from the nominal volume and must be used when calculating the effective fluid volume.

For diaphragm accumulators, the effective gas volume corresponds to the nominal volume.

4.1.4 Effective volume Volume of fluid which is available between the operating pressures p_{2} and p_{1} .

4.1.5 Permitted operating temperatures -10 °C ... +80 °C

standard design, others on request

4.1.6 Gas charge Hydraulic accumulators must only be charged with nitrogen. Never use other gases.

Risk of explosion!

In principle, only use nitrogen of at least Class 4.0 (filtration <3 µm). If other gases are to be used, please contact HYDAC for advice.

When supplied, the accumulator is only pre-charged for storage purposes. Higher pre-charge pressures are possible by arrangement.

4.1.7 Permitted pressure ratio Ratio of maximum operating pressure p2 to gas pre-charge pressure p₀. See catalogue section:

 HYDAC Accumulator Technology No. 3.000

4.1.8 General safety instructions On no account must any welding, soldering or mechanical work be carried out on the accumulator shell.

After the hydraulic line has been connected it must be completely vented. Work on systems with hydraulic dampers (repairs, connecting pressure gauges etc.) must only be carried out once the fluid pressbæge pdessærgasave been released.

The operating instructions must be observed!

- Bladder Accumulators No. 3.201.BA
- Diaphragm Accumulators No. 3.100.BA
- Piston Accumulators No. 3.301.BA

4.1.9 Working temperature and operating medium

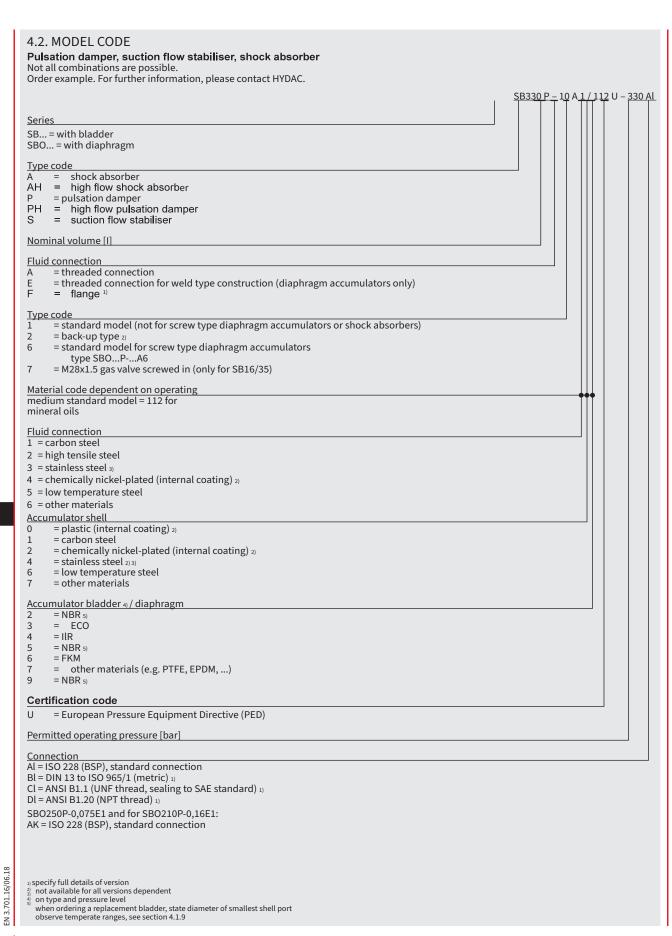
The permitted working temperature of a hydraulic damper is dependent on the application limits of the metal materials and the separation element. Outside this temperature range, special materials must be used. The operating medium must also be taken into account. The following table displays a selection of elastomer materials including max. temperature range and a rough overview of resistant and non-resistant fluids. Please contact us for help in selecting a suitable elastomer.

Materia	als	ort e		Temperature range	Overview of the fluids 2)	
		Materialcode 1)Accumulatort			Resistant to	Not resistant to
NBR	Acrylonitrile butadiene rubber	2	SB, SBO	-15 °C + 80 °C	Mineral oil (HL, HLP) Flame-resistant gfloutes from the	Aromatic hydrocarbonsChlorinated hydrocarbons
	rubber	5	SB, SBO	-50 °C + 50 °C	HFA, HFB, HFC Synthetic ● ester (HEES) Water	(HFD-S) ● Amines and ketones
		9	SB, SBO	-30 °C + 80 °C	• Sea water	Hydraulic fluids from the group HFD-R Fuels
ECO	Ethylene oxide epichlorohydrin rubber	3	SB	-30 °C +120 °C	Mineral oil (HL, HLP) Flame-resistant fluids from the group HFB Synthetic ester (HEES) Water	 Aromatic hydrocarbons Chlorinated hydrocarbons (HFD-S) Amines and ketones Hydraulic fluids from the group
			SBO	-40 °C +120 °C	• Sea water	HFD-R Flame-resistant fluids from the groups HFA and HFC Fuels
IIR	Butyl rubber	4	SB	-50 °C +100 °C	Hydraulic fluids from the group HFD-R	Mineral oils and mineral greasesSynthetic ester (HEES) Aliphatic,
			SBO	-50 °C +120 °C	Flame-resistant fluids from the Water	chlorinated and aromatic hydrocarbons Fuels
FKM	Fluorine rubber 6		SB, SBO	-10 °C +150 °C	Mineral oil (HL, HLP) Hydraulic fluids from the group HFD Synthetic ester (HEES)	Amines and ketonesAmmonia Skydrol andHyJet IV Steam
			● Fuels ● Aromatic hydrocarbons		• Fuels	

ı) see section 4.2. Model code, material code, accumulator bladder/diaphragm

2) others available on request



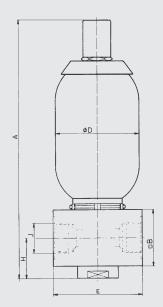




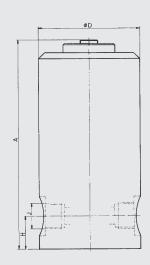
4.3. DIMENSIONS AND SPARE PARTS

4.3.1 Pulsation dampers bladder accumulator

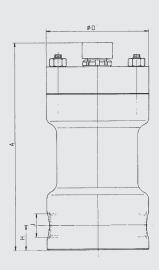
SB330/550P(PH)-...



SB800P-...



SB1000P-...

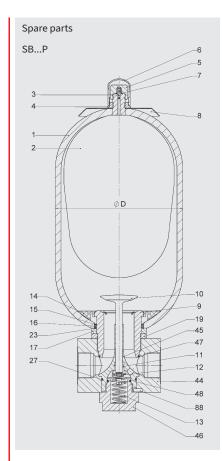


Dimensions SB

Nominal volume	Series	Max. operating pressure	Part no.		Eff. Gas	А	□В	ØD	Е	Н	Thread	Weight
[1]		(PED) [bar]	Carbon steel	Stainless steel	volume [l]	[mm]	[mm]	[mm]	[mm] [mm]		ISO 228	[kg]
1	SB330P	330	296114	3123952	1	365	80	118	120	57		11
1	SB550P	550	3435597	3132888		384	70	121	120	53	G 1 1/4	13
1.5	SB800P	800 3)			1.2	346	-	160 -		55		36
1.5	SB1000P	1000 3)	-	-	1.3	414	-	215	-	49	1)	94
2.5	SB330P	330	3078967	3108889	2.4	570	80 118			57		16
2.5	SB550P	550	3576155	3096755	2.5	589	70	121 12	0	53 G	1 1/4	20
	SB330P	220	3121155	3112225	2.7	455	80	171		57		18
4	SB330PH	330		_	3.7	491	100	171	150	85 G	1 1/2	26
5	SB550P	550	4313259	3136856	4.9	917	70	121	120	53	C 1 1/4	26
	SB330P		3140558	3251391	5.7	559	80	171	120	57	G 1 1/4	20
6	SB330PH	220	_	-	5.7	593 1	00	1/1		85	OF C11/2	28
10	SB330P	330	3082257	3114689	9.3	620				85	G 1 1/2	40
10	SB330PH			_	9.3	652	130x140			100	SAE 2" - 6000 psi	50
13	SB330P		2107871		12	712				85	C 1 1/2	48
20	SB330P	330	3084825	10)0 920- 18.4			229	150		G 1 1/2	70
	SB330PH		-	-		952	130x140] 229		100	SAE 2" - 6000 psi	80
24	SB330P		3152980 2	3.6 986-			100			0.5	C 1 1/2	82
32	SB330P	330	3121154	-	22.0	1445	100			85	G 1 1/2	100
32	SB330PH		-	_	33.9	1475	130x140			100 SA	E 2" - 6000 psi	110

¹⁾ M56x4, high pressure connection DN 16, others on request 2) standard connection code = AI, others on request 3) special design, on request





Description	Item I
Bladder assembly consisting of:	assen
Bladder	2
Gas valve insert*	3
Retaining nut	2 3 4
Seal cap	5
Protective cap	5
O-ring	7
Seal kit consisting of:	
O-ring	7
Washer	15
O-ring	16
Support ring	23
O-ring	27
O-ring	47
O-ring	48
* available separately	

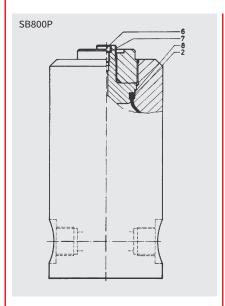
* available separately	
Accumulator shell (item 1) and	d company label (item 8) not
available as a spare part	

escription Connection	Item
ly consisting of:	
Oil valve body	9
Valve plate	10
Damping bush	11
Lock nut	12
Spring	13
Anti-extrusion ring*	14
Washer	15
O-ring	16
Spacer	17
Lock nut	19
Support ring (only for 330 bar)	23
O-ring	27
Connector	44
Guide piece	45
Cap	46
O-ring	47
O-ring	48
Locking key	88

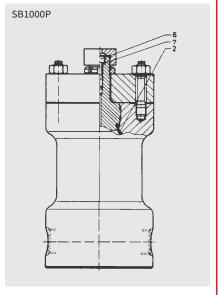
NBR, carbon steel Standard gas valve

Volume	Bladder assembly	Seal kit	
[[]		SB330P/SB400P	SB550P
1	237624		
2.5	236171		
4	236046	357055	2106402
5	240917		
6	2112097		
10	236088		
13	376249		
20	236089	357058	357061
24	376253		
32	235335		

* available separately



Description	Item
Bladder	2 E
Charging screw	6 (
Seal ring U 9.3x13.3x1	7 9
Support ring	8



	Description	Item
Е	ladder	2
C	harging screw	6
S	eal ring	7

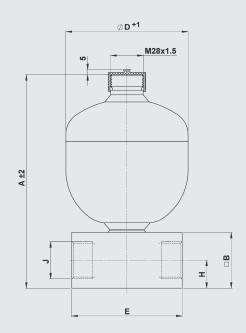
EN 3.701.16/06.18

86 HYDAC

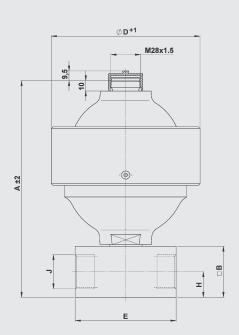


4.3.2 Pulsation dampers diaphragm accumulator

SBO...P...E (welded)



SBO...P...A6 (screwed)

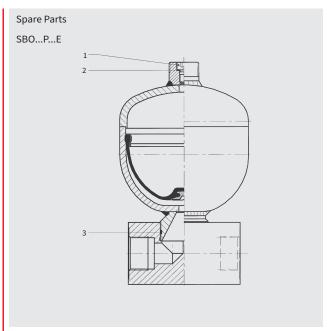


Dimensions SBO

Nominal volume	Series and connection type 1)	Max. opera pressure (P		А	□В	ØD	Е	Н	J Thread	Weight
[l]		Carbon steel [bar]	Stainless steel [bar]	[mm]	[mm] [n	nm]	[mm]	[mm]	ISO 228	[kg]
0.075	SBO250PE1AK	250	_	131	-	64	h 41	1.2	6.1/4	0.9
0.16	SBO210PE1AK		180	143	-	74	hex. 41	13	G 1/4	1
0.32	SBO210PE1Al	210	160	175	50	93	- 80	25	G 1/2	2.6
0.5	- SB0210PE1Al		-	192	50	105	7 80	25	G 1/2	3
0.6	SBO330PE1Al	330	_	222		115				5.6
0.75	SBO210PE1Al	210	140	217		121				5.1
1	SBO200PE1Al	200	-	231		136				6
	SBO140PE1Al	140	_	244		145				6.2
1.4	SBO210PE1Al	210	_	250		150				7.7
	SBO250PE1Al	250	_	255	60	153	105	30	G1	8.2
2	SBO100PE1Al	100	100	261		160				6.3
2	SBO210PE1Al	210	-	267	1	167				8.9
3.5	SBO250PE1Al	250	-	377		170]			13.5
4	SBO50PE1Al		50	368		158				7.9
4	SBO250PE1Al		180	377		170				13.5
0.25	SBO500PA6Al	500	350	162	50	115 (125)	80	- 25	G 1/2	5.2 (6.3)
0.6	SBO450PA6Al	450	250	202		140 (142)	95	25		8.9 (9.1)
1.3	SBO400PA6Al	400	-	267	1	199			1	13.8
2	SBO250PA6Al	250	180	285	60	201	105	20	G1	15.6
2.8	CDO 400D AC AL	400	-	308		252	105	30		24.6
4	SBO400PA6Al	400	_	325	1	287	1			36.6

 $_{1}$ standard connection code = AK or AI, others on request () brackets indicate different dimensions for stainless steel version



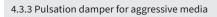


SBOPA6	
	1 2
	5
	4
	3

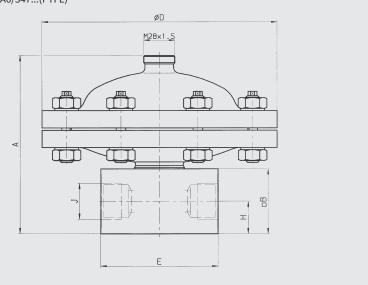
Item
1 Ch
2 Se
3 Se

Descriptions	Item
Charging screw	1
Seal ring	2
Seal ring	3
Diaphragm	4
Support ring	5





SBO...P-...A6/347...(PTFE)



Pulsation damper in stainless steel with PTFE-coated diaphragm. Also available without connection block.

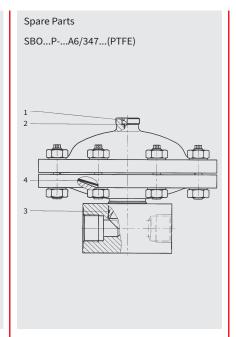
Permitted operating temperature: -15 $^{\circ}\text{C}$... +80 $^{\circ}\text{C}$

Permitted pressure ratio p_2 : $p_0 = 2:1$

Dimensions

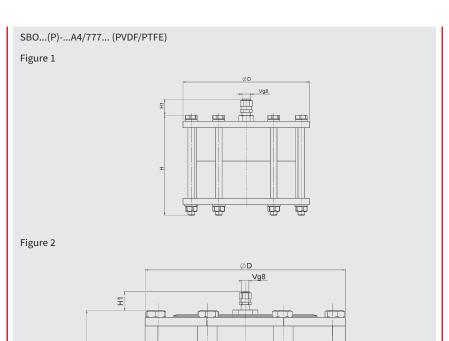
Nominal	Max.	Part no.	А	□BØ□)	E	Н	J 1)	Weight
volume	operating							Thread	
	pressure (PED)								
[1]	[bar]		[mm]	[mm] [ı	nm]	[mm]	[mm]	ISO 228	[kg]
0.2	40	4328332 19 1 40			210				11
0.2	250	4328333		60	230	105	30	C1	27
0.5	40	3091224	165	60	210	105	30	G1	12
	250	3091221	200		230]			26

1) Standard connection code = AI, others on request



Description Item	
Charging screw	1
Seal ring	2
Seal ring 3	
Diaphragm	4





Spare parts SBO...(P)-...A4/777... (PVDF/PTFE)

Descriptions Item Gas valve assembly 1 Gas valve insert brass / stainless steel 2 Diaphragm

Relevant operating instruction is available on request.

Pulsation damper in PVDF with PTFE-coated diaphragm.

_____Ø11

Permitted operating temperature: -10 °C ... +65 °C

Permitted pressure ratio p_2 : $p_0 = 2:1$

Dimensions

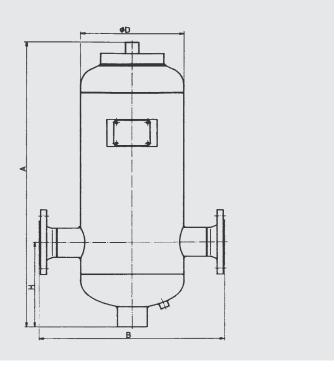
D							
Nominal volume	Max. operating pressure (PED)	Part no.	ØD	Н	H1	Weight	Figure
[l]	[bar]		[mm]	[mm]	[mm]	[kg]	
0.08	12	3655864 1	15	94	15	1.5	1
	10	-		128	20	5.7	
0.2	16	-		120	10	C 4	
	25	3357658 1	82	130	18	6.4	
	10	-		168	20	6	2
_	16	-		170	10	C 0	
	25	3357657]	170	19	6.8	

□158 4x∅11



4.3.4 Suction flow stabiliser

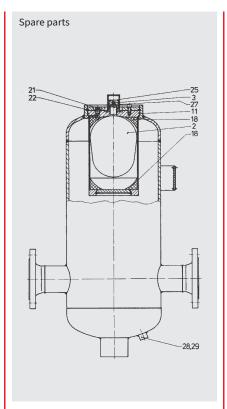
SB16S





SB16S – permitted operating pressure 16 bar (PED)										
Nominal volume	Fluid volume	Eff. gas volume	А	В	ØD	Н	DN*	Weight		
[l]	[1]	[l]	[mm]	[mm]	[mm]	[mm]		[kg]		
12	12	1	580	425	219	220	65	40		
25	25	2.5	1025	423	219	220	65	60		
40	40	4	890	540	300	250	80	85		
100	100	10	1150	650	406	350	100	140		
400	400	35	2050	870	559	400	125	380		

further pressure ranges 25 bar, 40 bar, others on request other fluid volumes on request

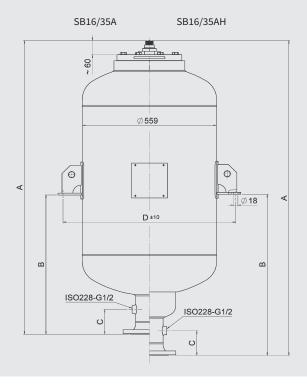


Description	Item
Bladder	2
Gas valve insert	3
O-ring	11
Insertion ring, 2x	18
Screw plug	21
Retainer ring	22
Cap nut	25
O-ring	27
Seal ring	28
Screw plug	29

^{*} to EN1092-1/11 /B1/PN16



4.3.5 Shock absorber



Dimensions

SB16/35A

perm. operating pressure 16/35 bar (PED)

N]bEnffinal		SB16A	BB16A							SB35A					
volume	gas volu	nReirt no.		[mm] B		С	D <u>+</u> 10	Weight Pa	art no.		А	В	С	D ±10	Weight
	Carbon		Stainless (approx.) (a	pprox.) (a	prox.)		Carbon		Stainless (approx.) (a	pprox.) (ap	prox.)		_
	[l]	steel	steel		[mm] [mn	n] 493	[mm]	[kg]	steel steel	[mm]	[mm]		[mm]	[mm]	[kg]
150	149	4108288	41082411	044				127	4108339	41083061	076	578			171
200	203	4108290	4093557 1	275	691			149	41083414	108307 131	8	699			208
300	288 4108	291	4108242 1	644	920	108	720 17	8	41083424	108308 170	1	937	121	728	261
375	374	4108292 4	108243 202	0	1063			214	41083554	108312 208	6	1083			315
450	453	4108294	4108244 2	361	1234			244	41083574	108314 243	6	1258			364

Flange to EN1092-1/11 / DN100 / PN16 or PN40 others on request

SB16/35AH perm. operating pressure 16/35 bar (PED)

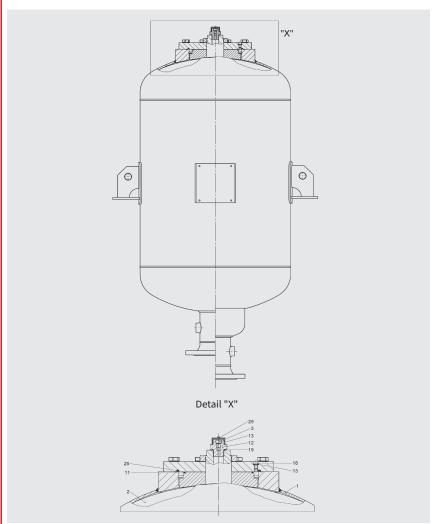
	The state of the s														
Nominal	Eff.	SB16AH							SB35AH						
volume	gas	Part no.		Α	В	С	D ±10 \	Veight	Part no.		Α	В	С	D ±10	Weight
	volume	Carbon	Stainless	(approx.)	approx.) (approx.)		-	Carbon	Stainless (approx.) (a	pprox.) (ap	prox.)		_
[l]	[1]	steel	steel	[mm]	[mm]	[mm]	[mm] 1	3[\$kg]	steel	s[trenerh] 410	8729	[mm]	[mm]	[mm]	[kg]
150	149	4108720	4108702 1	135	638				4108773	1166		641			180
200	203	4108721	4108703 1	366	754			157	4108775	4108730 1	408	762			217
300	288	41087244	108715 173	5	988	108	720 18	6	4108774 4	108734 179	1	1000	121	728 27	0
375	374	4108725 4	108717	2111	1127			222	4108776 4	108758 217	6	1146			324
450	453	4108726	4108718 2	452	1298			252	4108778 4	108762 252	6	1321			373

Flange to EN1092-1/11 / DN100 / PN16 or PN40 others on request



Spare parts

SB16/35A, SB16/35AH
The following spare parts relate exclusively to hydraulic accumulators from construction year 2016 and later.
For low-pressure accumulators of series SB16/35A and SB16/35AH up to construction year 2015 spare parts are available on request.



Description	Item
Bladder	2
Gas valve assembly consisting of:	
Screw plug	3
Gas valve body	12
Seal ring	13
O-ring	19
Protective cap	29
Seal kit consisting of:	
O-ring	11
Seal ring	13
Vent screw	18
O-ring	19
O-ring	25

Accumulator shell (item 1) not available as a spare part

NBR

NDIC											
Nominal B volume [l]	adder	Seal kit									
votume [t]		Carbon steel	Stainless steel								
150	4241264										
200	4241263										
300	4113771	4241465	4197141								
375	4113731										
450	4241435										